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Abstract: Splash zone crossing of the structures with large horizontal surface (e.g. manifolds) and the structures having large weight variation in water and air (e.g. suction anchors) is a critical marine operation. This is due to the large slamming forces and added mass of the structure, which results in high dynamic loads on the crane. The solution to this could be attaching a PHC (Passive Heave Compensator) between the crane hook and the payload. This paper analyzes the deployment of a subsea manifold with and without PHC unit in North Sea at a water depth of approximately 370 m. A detailed dynamic analysis is done for a seastate of 3 m significant wave height (Hs) over a range of zero up-crossing period (Tz) varying from 3s to 13s. For better understanding of the result analysis has been done in two stages. The first stage covers the lowering of manifold through the splash zone while second stage covers the seabed landing of the manifold. Based on the results of the analyses it is concluded that PHC tends to reduce the dynamic peak load on the crane. Besides this, it also mitigates the risk of slack wire situations during splash zone crossing of the payload. Furthermore, reduction in both landing velocity and crane tip velocity is also achieved by using a well-designed PHC unit.

Key words: PHC, dynamic analysis, Orcaflex.

1. Introduction

During the development of an offshore oil and gas field, marine operations play a very important role. One of the vital marine operations is offshore lifting of a structure, which consists of various operational phases: lifting in the air, lowering through the splash zone, lowering down to seabed and seabed landing [1]. Amongst the aforementioned phases, lowering of structures through splash zone is the most critical operational phase. This is because it is during this phase (i.e. splash zone crossing) where the maximum forces and lowest weather restrictions are expected to be found. Therefore, design loads must also be established for this phase of the operation [1].

While assessing the crane design loads, the dynamic loads due to operational motion must be accounted for. This is achieved by multiplying the working/static load by a dynamic factor (ψ), which takes into account the inertia forces and shock [2]. Also due to added

dynamics in the rough sea-state, the working load of an offshore crane depends on the significant wave height (Hs). For e.g. a typical crane SWL (Safe Working Load) is reduced by 30 % at 2 m Hs and by 50% at 3 m Hs [3].

To reduce the dependency of working load of the crane on Hs and to increase its working load capacity, a PHC (Passive Heave Compensation) unit is attached between the crane hook and the payload. This manuscript briefly explains PHC and its impact on offshore lifting by performing dynamic analysis on Orcaflex. Section 2 of the paper defines PHC and briefly discusses the working principle, efficiency and application of the PHC. Thereafter, in section 3 a case study demonstrating the installation analyses of subsea manifold using Orcaflex has been done. Finally, a suitable conclusion is presented in section 4.

2. Passive Heave Compensator

2.1 Definition

PHC is "an offshore equipment, generally connected

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between the crane hook and the payload, such that it stores the energy from waves influencing the payload and dissipate it later" [3]. A general PHC unit consists of a spring-damper system constructed with cylinder, piston, nozzles and accumulator as depicted in Fig. 1 [4]. The cylinder is filled with hydraulic oil, while the accumulator is filled with Nitrogen. These units have a certain available stroke limit, which should not exceed during the lifting operation. If exceeded, it may cause large peak loads thereby causing failure of the hoisting system [5].

2.2 Working Principle

For a crane mounted on the floating vessel the dynamic loads due to operational motion are taken into consideration by multiplying the working load by a dynamic factor (ψ). The dynamic factor (also called Dynamic Amplification Factor—DAF) takes into account the inertia force and shock and is given by Ref. [2]:

$$\psi = 1 + V_r * \sqrt{\left(\frac{C}{W * g}\right)}$$

The load on the crane wire during offshore lifting operation is equal to the dynamic load (F_d) , which is equal to $\psi * W$. The dynamic load (F_d) can be reduced by following ways:

Reduce working/static load (*W*).

Reduce *Vr*, which implies waiting for lower waves or working in the lower wave condition.

Reduce C.

If we intend to reduce F_d by reducing option 1 and 2; then we are compromising on productivity. Alternatively, by providing a soft link or a device having low stiffness between the crane hook and the payload, reduction in *C* is achieved, which ultimately leads to mitigation of F_d . This forms the bases of PHC whose working principle is explained next.

The PHC is in principle a pure spring damper system, which does not require input of energy during operation [5].As shown in Fig. 1, the payload is attached to lower end of the piston rod, which causes it to extend. As the piston rod extends, it forces the oil in the cylinder to flow into the accumulator via nozzle. The nozzle restricts the flow and provide necessary dampening effect, while the gas that is being compressed by upward motion of piston in accumulator, provides the spring effect. Thus, the combination of spring and dampening effect isolates the payload from the wave motion and provides the required heave compensation [6].

The stiffness of the compensator is proportional to the gas pressure inside the accumulator, which varies with the motion of the piston [7]. Moreover, the hydraulic dampening force for PHC is given by Ref. [2]:

$$F_d = A_k^3 * \frac{\rho}{2} * (\frac{1}{\alpha * A} * S_v)^2$$

The aforementioned equation depicts that F_d is proportional to square of stroking velocity. Hence, increase in stroke velocity, which refers to increased heave motion increases F_d as well. However, it must be noted that dampening and stiffness characteristics can be changed to suit different lifts. This is achieved by changing nozzles, oil level or accumulator pressure [8].

2.3 Efficiency

The ratio between the response of the lifted object and the excited motion is expressed through a complex transfer function [5]:



Fig. 1 Schematic of typical PHC.

$$\frac{\eta_3}{\eta_{3T}} = G(\omega)$$

Efficiency of the PHC in terms of the complex transfer function is given as:

 $e = 1 - |G(\omega)|$

Where $G(\omega)$ is transmissibility and *e* is efficiency of the PHC, which tells us whether PHC will contribute positively during offshore lifting operation or not. Furthermore, based on the efficiency formulae it is stated that efficient heave compensation is obtained when:

The natural frequency, ωo is as low as possible.

Both drag forces and added mass are large.

Stiffness of the PHC is low.

Heave compensator damping is low enough to avoid resonance.

Both wave period and mass density of the payload are low as shown in Fig. 2.

2.4 Offshore Application

During the offshore lifting operation, PHC is connected between the crane hook and the payload in order to reduce the dynamic load on the hoisting system and the crane tip. Some of its application areas are:

(a) **Splash zone lifts:** As the AHC (Active Heave Compensator) does not function well in the splash zone,

so PHC is used as the compensating device. The PHC absorbs the huge dynamic forces on payload during splash zone crossing, thus reducing the dynamic loads on the crane tip and the hoisting system. The PHC also reduces the chances of slack wire during splash zone lifting; nevertheless, if the slack occurs in the wire then the piston rod extends to compensate this removal of slack and absorbs the snap load resulting from this tautening process.

(b) Seabed landing: The PHC is used to attain reduction in landing velocity of the structure on the seabed. This is important because generally the landing velocity of the payload is very close to the hoisting velocity (Vc) of the winch, which is assumed to be 0.5 m/s, if it's value is unknown [5]. Landing at such high velocities causes damage to the seabed structure. Furthermore, the structure may have tendency to rebound on hitting the seabed, which may damage the crane. Both of these detrimental effects are avoided by using well designed PHC unit.

(c) **Resonance avoidance:** During offshore lifting the crane wire stiffness changes with the water depth, which leads to the change in frequency of the hoisting and payload system. If during this lowering process the period of crane tip movement matches with the frequency of the hoisting and payload system, resonance may occur. Such a situation must be avoided,



Fig. 2 Efficiency of typical PHC[3].

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as resonance leads to large dynamic loads on the entire system, which in the worst case may cause failure of the hoisting system. However, resonance is avoided by adding a PHC between the crane hook and the payload, as PHC increases the system frequency to a level which cannot be matched by the wave frequencies [3].

3. Illustrative Case Study

3.1 Purpose and Scope

The purpose of the analysis is to compare the offshore lifting operations with and without PHC; thereby proving that by using a PHC between the crane hook and payload, dynamic load on the hoisting system and the crane is reduced. Furthermore, it is intended to show that crane tip velocities and landing velocity of the payload are also reduced by the use of PHC.

The analysis is done using Orcaflex to assess the operators viability in the Hs of 3 m over a range of Tz varying from 3 s to 13 s and using the methodology mentioned in Ref. [5]. The crane wire hoisting velocity Vc of 0.5 m/s Ref. [5] is used in the analysis and vessel heading is assumed to be restricted within 15 degrees of head sea. For the sake of simplicity, the analysis is divided into two phases:

(a) First phase—lowering through splash zone: In this part of the operation the governing forces and lowest weather conditions are expected to be found. Hence, this part of operation is very critical from design point of view. Analysis is done to check:

(1) Maximum and minimum crane wire tension (with and without PHC).

(2) Maximum and minimum lifting slings tension (with and without PHC).

(3) Identifying slack in lifting slings (with and without PHC).

(b) Second phase—landing on seabed: The crane wire tension is expected to be lower during this part of operation as compared to lowering through splash zone. During this phase the utility of PHC is depicted as a device which reduces the landing velocity of structure on seabed. Analysis is done to check:

(1) Maximum and minimum crane wire tension (with and without PHC).

(2) Maximum crane hook velocities (with and without PHC).

The maximum and minimum value of tension and velocity is found using a Gumbel distribution.

3.2 Environmental Conditions

For installation analysis of the manifold, an assumed site in the North Sea in water depth of 370 km is chosen. The JONSWAP wave spectrum is used in the analysis and only one wave direction is considered in the analysis. During the analysis it is assumed that the vessel is free to weather vane during all operations except for the landing operation. Furthermore, the analysis is run for head seas (0 degrees) ± 15 degrees (-15 degrees are waves coming from starboard and ± 15 degrees are waves coming from port as shown in Fig. 3). Hence, the analysis is done for the wave directions 165 degree, 180 degree and 195 degree.

The analysis does not include shielding or refraction of waves. Therefore, results for waves coming from -15 degrees are thus expected to be conservative, because in real life the vessel has a shielding effect, which calms the water surface in the area, where the template crosses the water surface. Furthermore, short term wave condition as defined in Ref. [5] is used and the analysis is performed for Hs = 3 m and Tz varying from 3 s to 13 s.

3.3 Analysis Model

The model for our case study is comprised of the IV (installation vessel), a subsea manifold (payload), four lifting slings, crane winch wire and the PHC. The Orcaflex model of IV and associated information about loading condition and displacement RAOs are taken from one of the engineering contractors. Since, the weight of the manifold is less than 1% of the total mass of the vessel, therefore, it is assumed that manifold has very less influence on the vessels motion.



Fig. 3 Definition of wave direction.



Fig. 4 Orcaflex manifold model.

The manifold is a complex structure consisting of many parts with varying hydrodynamic properties. The dimensions of manifold are assumed 12m * 6m * 7m, with assumed mass of 90 tons and submerged weight of 78 tons. For the modeling process in Orcaflex, the manifold is divided into various parts as shown in Fig. 4 and listed below:

(a) The main body where all piping, valves etc. are present (80 tons).

- (b) The roof (8 tons).
- (c) 4 roof support structures (0.5 tons each).

It is assumed in the analysis that the main body and the roof are hydrodynamically independent to each other. Furthermore, based on guidance note given in Ref. [5] it is assumed that the roof support structures does not contribute to any vertical hydrodynamic forces as they are in the shadow of the roof and the main body.

The Orcaflex 3-D model of the PHC is depicted in Fig. 5.



Fig. 5 Orcaflex PHC model.

The entire model of the PHC along with the associated stiffness and damping values are taken from the crane master calculation sheets.

3.4 Result

A substantially large number of simulation runs are performed for each sea state in order to obtain realistic amplitudes on the peak forces. For our case, a 3-hoursimulation run is done using a pre-processing excel spreadsheet provided in the Orcaflex software. After this post-processing of results is done using Gumbel distribution for a PNE (probability of non-exceedance) of 95% using the post-processing, excel spreadsheet provided in the Orcaflex software.

The excel sheets used for pre-processing and post-processing during analysis were taken from Ref. [9], and as a reference are provided in Appendix. Finally, the results are presented separately for the two phases of lifting operation. However, for both the operations, dynamic simulation simulates crane wire pay-out, whose lowering velocity is set to 0.5 m/s [5].

3.4.1 Splash Zone Result

The analysis starts with the manifold hanging

completely in air and then being lowered in air, followed by lowering through the splash zone as depicted in Fig. 6. The simulation continues until the entire manifold is submerged completely in the water.

The results are summarized in Table 1 and 2, with the maximum values highlighted by red colour.

By comparing the values in Table 1 and 2 it is inferred that the PHC leads to reduction of maximum crane wire tension during the splash zone crossing. The same trend is followed by the slings. However, it is clearly seen that the variation in crane wire tension for longer Tz periods is smaller when compared to smaller Tz periods. This indicates that the efficiency of PHC decreases for longer Tz periods as depicted in Fig. 2. Hence, it is beneficial to use PHC only for low to medium Tz periods. The comparative results are also presented in the time series graph shown in Fig. 7.

Fig. 7 depicts that during lowering operation in the air, only the dry weight of the manifold and forces due to crane tip accelerations are acting on the structure. Due to cancelation of stiffness and dampening effect of the PHC by its weight the crane wire tension for both the cases is nearly the same for this phase. However,



Fig. 6 Wireframe model for lowering through splash zone.

Without PHC	: Splash Zone											
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Max Crane Wire Tension [kN]	2108.22	2033.46	2113	1913.2	1530.4	1496.6	1255.4	1130.4	1073.3	1099.3	1051.01
	Max Sling#1 Tension[kN]	904.798	626.991	650.668	607.49	480.58	458.38	376.29	352.01	343.58	356.59	329.043
165deg	Max Sling#2 Tension[kN]	853.265	714.754	648.322	564.53	477.04	434.97	359.19	325.66	308.93	311.5	289.279
	Max Sling#3 Tension[kN]	840.533	613.786	787.226	584.43	463.44	458.53	404.47	349.73	333.59	339.58	328.96
	Max Sling#4 Tension[kN]	916.966	684.142	653.25	596.04	469.53	446.35	382.41	333.7	317.3	336.11	301.515
	Tz (s)	3	4	5	б	7	8	9	10	11	12	13
	Max Crane Wire Tension [kN]	2446.88	2075.53	1998.39	1851.6	1481.1	1408.3	1228.2	1141.2	1069.5	1055.4	1003.08
180deg	Max Sling#1 Tension[kN]	788.047	759.053	656.572	562.15	488.66	434.31	362.48	343.03	334.02	333.24	319.075
	Max Sling#2 Tension[kN]	860.181	739.558	648.692	582.87	463.94	405.17	358.09	327.76	309.22	290.47	284.558
	Max Sling#3 Tension[kN]	791.772	690.461	688.558	538.33	447.42	446.42	388.11	361.39	334	321.2	319.977
	Max Sling#4 Tension[kN]	837.303	751.136	586.045	546.28	421.64	409.88	379.86	343.98	321.46	322.42	308.106
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Max Crane Wire Tension [kN]	2183.68	2189.79	1761.38	1521.7	1388	1267.2	1191.9	1187.3	992.96	1036.2	991.844
	Max Sling#1 Tension[kN]	755.08	718.967	535.061	484.95	417.76	404.83	382.92	370.94	310.9	329.83	310.851
195deg	Max Sling#2 Tension[kN]	863.198	731.064	539.398	474.35	416.41	385.32	351.85	342.9	282.43	281.97	274.787
	Max Sling#3 Tension[kN]	762.68	771.99	568.62	466	404.3	418.8	389.9	352.7	315.4	330.5	317.31
	Max Sling#4 Tension[kN]	879.35	767.72	580.8	479.1	432	390.7	358.7	357.5	302.3	311	292.16

Table 1Splash zone crossing result without PHC (Hs = 3 m).

Table 2Splash zone crossing result with PHC (Hs = 3 m).

With PHC: Splash Zone												
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Max Crane Wire Tension [kN]	1586.7	1346.1	1685.8	1313.9	1202.4	1177	1085.1	1058.6	1015.3	1006.3	970.6
	Max Sling#1 Tension[kN]	537.2	463.8	614	469.9	396.9	389.1	363.1	337.6	323.8	325.6	312.4
165de g	Max Sling#2 Tension[kN]	469.232	387.35	539.51	366.5	339.627	321.91	303.711	299.49	276.839	280.83	265.509
	Max Sling#3 Tension[kN]	536.605	444.48	597.22	444.95	383.212	367.72	364.34	346.33	325.642	330.45	319.234
	Max Sling#4 Tension[kN]	498.534	410.64	557.71	395.13	365.311	357.35	330.57	324.03	302.162	310.13	298.719
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Max Crane Wire Tension [kN]	1486.4	1384.9	1733.2	1311.5	1063.9	1094.1	1021	998.1	949.4	938.2	954.9
	Max Sling#1 Tension[kN]	514.8	488.7	587.1	498.3	373.2	352.6	336.8	323.5	308.2	303.2	308
180deg	Max Sling#2 Tension[kN]	471.85	422.72	514.2	396.95	306.662	300.09	285.865	275.21	262.23	258.29	262.479
	Max Sling#3 Tension[kN]	484.728	488.54	582.14	475.4	355.63	362.24	329.424	323.03	304.586	302.52	307.479
	Max Sling#4 Tension[kN]	481.639	457.62	519.15	416.6	348.632	334.84	304.791	303.13	285.099	283.74	287.727
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Max Crane Wire Tension [kN]	1497.21	1248	1446.7	1253.9	1056.93	1146.8	1058.1	1005.3	982.454	967.42	984.152
	Max Sling#1 Tension[kN]	478.5	416.3	495.3	423.9	367.2	374.9	340.7	325.3	318.6	314.4	319.2
195deg	Max Sling#2 Tension[kN]	450.078	375.79	410.21	356.18	325.182	328.87	290.672	280.63	275.135	266.79	268.777
	Max Sling#3 Tension[kN]	480.01	412.1	489.8	422.4	356.87	369.9	348.69	325	316.72	313.5	322.96
	Max Sling#4 Tension[kN]	460.34	383.4	443.3	393.8	334.03	345.4	322.32	305.6	303.21	292	299.16



Fig. 7 Time history graph showing crane wire tension with and without PHC (Hs = 3 m).

without PHC: Splasn Zone												
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Minimum Sling#1 Tension[kN]	-29.9	-33.4	-17.0	21.4	51.9	95.6	106.2	143.4	158.7	161.5	168.4
165de g	Minimum Sling#2 Tension[kN]	-18.5	-20.4	-13.5	14.8	30.0	75.2	82.3	98.2	120.4	113.3	123.6
	Minimum Sling#3 Tension[kN]	-28.3	-31.9	-14.7	31.9	53.2	100.4	114.1	136.6	163.3	162.6	168.8
	Minimum Sling#4 Tension[kN]	-26.1	-28.0	-13.4	4.0	50.5	87.5	86.4	121.4	141.4	138.8	151.1
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Minimum Sling#1 Tension[kN]	-28.3	-28.0	-36.9	15.9	73.2	89.1	109.8	155.6	170.0	174.1	184.6
180deg	Minimum Sling#2 Tension[kN]	-18.2	-13.0	-23.5	-4.7	66.7	59.3	62.2	113.0	135.6	133.0	138.8
	Minimum Sling#3 Tension[kN]	-17.8	-18.3	-28.1	-10.0	82.6	86.8	112.5	155.4	169.8	178.5	181.9
	Minimum Sling#4 Tension[kN]	-12.0	-19.5	-32.0	-3.0	68.2	79.8	92.1	133.7	150.0	158.5	163.2
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Minimum Sling#1 Tension[kN]	-24.8	-35.2	-7.1	8.3	107.1	111.2	136.7	158.5	158.0	189.6	168.8
195de g	Minimum Sling#2 Tension[kN]	-13.0	-20.6	8.7	3.6	64.6	75.5	105.2	117.3	115.6	142.9	136.0
	Minimum Sling#3 Tension[kN]	-22.0	-33.5	-12.1	24.0	109.9	118.5	138.9	156.5	161.7	191.1	181.2
	Minimum Sling#4 Tension[kN]	-17.4	-27.1	-4.2	24.6	95.9	96.6	123.0	141.3	137.3	178.0	151.7

Table 3 Splash zone crossing result for slings without PHC (Hs = 3 m).

during the splash zone lowering, manifold experiences the highest force variations due to the transient hydrodynamic effects. Thus, the time series shows quick changes in crane wire tension. In this zone, PHC reduces both the peak loads and fluctuations in crane wire tension, thereby rendering its utility. However, as soon as the manifold is fully submerged in sea, the time series becomes smaller indicating that the mean force in the crane wire is reduced due to the buoyancy of the structure.

The same trend is observed in the time series of the lifting sling forces, however, the slings experience slack. The slack criteria used for analysis is $F_{hyd} \le 0.9 * F_{mini.static}$ [4]. As the assumed submerged weight ($F_{min.static}$) of module is 765 kN (78 tons), so by using the above criteria, if the minimum tension in the

slings is below 10% of 765 kN (i.e. 76.5 kN) the slings are assumed to be slacked. The slacked slings are shown as green blocks in Tables 3 and 4.

Table 3 indicates a lot of slacked slings for lower to medium Tz periods. It is recommended that slacks are avoided during splash zone crossing of the manifold, as slacked slings cause huge snap forces on the hoisting system and crane tip. Tables 3 and 4 depict that PHC leads to complete reduction in slack wires for Tz period of 3 s and 4 s; hence indicating its high efficiency at low periods. However, the PHC has led to slacking of wire at higher periods. This is not an issue, as the piston rod of the PHC extends to compensate this removal of the slack and absorbs the snap load resulting from the tautening process, thereby protecting the hoisting system from huge snap forces.

with FHC. spiash Zone												
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Minimum Sling#1 Tension[kN]	138.7	129.8	-33.7	12.9	57.2	27.3	32.3	60.3	81.1	72.6	125.6
165deg	Minimum Sling#2 Tension[kN]	103.4	95.1	-43.9	-3.6	35.0	6.4	15.7	34.3	63.3	44.5	90.8
	Minimum Sling#3 Tension[kN]	144.5	132.4	-36.5	9.6	49.4	20.3	25.2	52.2	73.5	65.3	109.4
	Minimum Sling#4 Tension[kN]	128.1	128.0	-23.2	11.2	52.5	25.1	28.6	52.4	82.8	63.2	103.3
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Minimum Sling#1 Tension[kN]	161.3	128.4	4.3	26.3	103.4	94.3	127.4	144.7	188.5	158.0	200.5
180deg	Minimum Sling#2 Tension[kN]	121.7	86.8	-10.9	28.1	74.6	65.1	95.6	127.8	149.4	122.6	159.4
	Minimum Sling#3 Tension[kN]	164.1	143.9	9.8	19.3	94.2	88.5	122.1	138.9	187.7	138.7	196.9
	Minimum Sling#4 Tension[kN]	149.4	124.2	8.6	44.5	90.6	85.4	116.3	154.1	175.0	130.8	183.8
	Tz (s)	3	4	5	6	7	8	9	10	11	12	13
	Minimum Sling#1 Tension[kN]	162.4	149.0	86.0	43.7	74.2	43.8	74.5	91.5	112.9	124.4	122.0
195deg	Minimum Sling#2 Tension[kN]	103.5	100.6	54.1	26.7	45.6	19.2	46.1	62.1	74.6	91.1	90.1
	Minimum Sling#3 Tension[kN]	161.1	168.4	58.7	34.9	65.3	36.2	50.3	80.5	108.1	118.5	116.2
	Minimum Sling#4 Tension[kN]	137.5	148.2	62.2	36.8	60.9	37.7	49.6	77.2	97.4	111.2	110.6

Table 4 Splash zone crossing result for slings with PHC (Hs = 3 m).

Without PHC: Seabed Landing												
	Tz	3	4	5	6	7	8	9	10	11	12	13
165deg	Max Crane Wire Tension [kN]	770.94	963.62	1230	1098.2	1168.2	1228.8	968.7	1001.3	1048.5	955.2	1020.3
	Max Crane Tip Velocity [m/s]	0.8252	2.9514	4.034	3.484	2.3176	2.8277	2.26	1.6884	1.7452	1.695	2.0857
	Tz	3	4	5	6	7	8	9	10	11	12	13
180deg	Max Crane Wire Tension [kN]	7 6 2.25	889.39	1261	1068.5	1135.1	1113.4	958	929.61	902.27	887.5	936.3
	Max Crane Tip Velocity [m/s]	0.8098	2.1271	2.844	2.4553	2.031	2.3911	1.802	1.3984	1.5303	1.496	1.6105
	Tz	3	4	5	6	7	8	9	10	11	12	13
195deg	Max Crane Wire Tension [kN]	761.79	895.37	970.9	1005.2	1067.9	1096.7	1018	995.36	887.31	912.3	970.28
	Max Crane Tip Velocity [m/s]	0.8152	1.4476	1.941	1.8372	2.0516	2.0585	1.692	1.6777	1.3431	1.513	1.583

With PHC: Seabed Landing												
	Tz	3	4	5	6	7	8	9	10	11	12	13
165deg	Max Crane Wire Tension [kN]	778.35	805.01	871.1	926.55	1041.5	1154.6	1075.4	1074.2	1110.7	1060.7	1154.1
	Max Crane Tip Velocity [m/s]	0.60561	0.6655	0.8235	0.8312	1.7551	2.833	2.7831	2.3799	2.1275	1.8926	2.2877
	Tz	3	4	5	6	7	8	9	10	11	12	13
180de g	Max Crane Wire Tension [kN]	778.15	793.61	848.22	928.38	1071.1	1149.8	1061.1	1085.1	1017.4	969	1044.9
	Max Crane Tip Velocity [m/s]	0.6065	0.6501	0.7398	0.8721	1.5476	2.1886	2.1362	1.8868	1.8904	1.6189	1.9381
	Tz	3	4	5	6	7	8	9	10	11	12	13
195deg	Max Crane Wire Tension [kN]	778.599	790.74	828.47	991.43	1190.7	1211.8	1145.5	1137.7	991.25	1060.4	1098.7
	Max Crane Tip Velocity [m/s]	0.61257	0.642	0.6914	1.0844	2.1816	2.7569	2.3948	2.1726	1.4972	1.9216	2.2989

Table 6Result for seabed landing with PHC (Hs = 3 m).



Fig. 8 Time history graph showing seabed landing velocity with and without PHC (Hs = 3 m).

3.4.2 Seabed Landing Result

During this phase of operation apart from maximum crane wire tension we are also interested to know the maximum crane tip velocity and the landing velocity of manifold on the seabed. The results are given in Tables 5 and 6: On comparing maximum crane wire tension of Tables 1 and 5 we find that values in Table 5 are much smaller than values in Table 1 (hence also smaller than MBL). This indicates that crane wires and other lifting slings must be designed for splash zone operation.

In addition, the main reason of using PHC for seabed

landing is to reduce the landing velocity of the structure on the seabed; which ultimately leads to reduction in the crane tip motions and velocities. On comparing the maximum crane tip velocity from Tables 5 and 6, we can deduce that for lower Tz values PHC reduces the maximum crane tip velocity. This indicates that PHC has higher efficiency at lower periods. Fig. 8 compares the seabed landing velocity of the manifold with and without PHC.

Fig. 8 depicts that during seabed landing of the manifold without PHC, the fluctuations in the landing velocities are large as compared to the landing velocity with PHC. Hence, for this phase of offshore lifting PHC reduces both, the maximum landing velocity and variations in the velocity by keeping payload at constant velocity thereby rendering its utility.

4. Conclusion

Based on the discussion of results in section 3 of this manuscript various conclusions are drawn:

(a) During offshore lifting operation, the maximum tension in crane wire and slings occurs for splash zone. Also during this phase the payload and hoisting system experiences the highest force variations due to transient hydrodynamic effects.

(b) PHC leads to reduction in tensions in crane wire and lifting slings. In other words PHC reduces dynamic loads on the hoisting system.

(c) The chances of slack slings is reduced by the use of PHC. However, even if slack slings occur with PHC the piston rod of PHC extends to compensate this removal of slack. Furthermore, PHC absorbs the snap load resulting from the sling tautening process; thereby protecting the hoisting system from huge snap forces.

(d) For the seabed landing operation PHC leads to reduction in maximum crane tip (heaving) velocities and landing speed of structure. PHC also reduces the variations in landing velocity by keeping the payload at constant velocity, hence abstaining hoisting system from huge dynamic accelerations and forces.

(e) PHC has highest efficiency at low Tz periods and with increasing wave period PHC becomes less efficient.

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FOR 165 D	EGREE; Hs=	3m									
Crane Wire Tension [max. kN]	Crane Wire Tension [min. kN]	Delete Max.	Delete Min.	Fltd. Max Crane Wire	Fltd. Min Crane Wire	Statistica Proces	stical Post- rocessing Gur		nble	filename;	Tz
1423.2	244.7	1423.2	244.7	1423.2	244.7	Raw Max	3388.6	Max		Case_A_01.dat	3
2054.5	7.8	2054.5	7.8	2054.5	7.8	Raw Min	0.0	Median	1591.7	Case_A_02.dat	
3388.6	0.0					Fltd Max	2054.5	StdDev	254.4	Case_A_03.dat	
2130.5	0.0	2130.5				Fltd Min	0.4	Beta	198.4	Case_A_04.dat	
1591.7	1.8	1591.7	1.8	1591.7	1.8	Ave Max	1684.6	Mode, µ	1519.0	Case_A_05.dat	
1918.0	0.4	1918.0	0.4	1918.0	0.4	Ave Min	56.1	Ex Max.	2108.2	Case_A_06.dat	
1857.5	2.0	1857.5	2.0	1857.5	2.0			Min		Case_A_07.dat	
2229.3	0.0	2229.3						Median	3.9	Case_A_08.dat	
1866.7	0.0	1866.7				PE	95 %	StdDev	96.0	Case_A_09.dat	
2337.2	0.0	2337.2						Beta	74.9	Case_A_010.dat	
1496.6	3.9	1496.6	3.9	1496.6	3.9			Mode, µ	-23.5	Case_A_011.dat	
1450.4	132.0	1450.4	132.0	1450.4	132.0		Ex	pected Min.	-105.7	Case_A_012.dat	
1478.3	6.1	1478.3	6.1	1478.3	6.1	Raw Max	1992.0	Max		Case_A_013.dat	4
1437.7	101.4	1437.7	101.4	1437.7	101.4	Raw Min	0.6	Median	1608.8	Case_A_014.dat	
1212.2	80.8	1212.2	80.8	1212.2	80.8	Fltd Max	1856.0	StdDev	209.2	Case_A_015.dat	
1856.0	0.9	1856.0	0.9	1856.0	0.9	Fltd Min	0.9	Beta	163.1	Case_A_016.dat	
1650.5	6.2	1650.5	6.2	1650.5	6.2	Ave Max	1586.3	Mode, µ	1549.0	Case_A_017.dat	
1598.3	3.6	1598.3	3.6	1598.3	3.6	Ave Min	58.0	Ex Max.	2033.5	Case_A_018.dat	
1784.3	5.0	1784.3	5.0	1784.3	5.0			Min		Case_A_019.dat	
1992.0	0.6							Median	7.9	Case_A_020.dat	
1608.8	210.7	1608.8	210.7	1608.8	210.7	PE	95 %	StdDev	82.0	Case_A_021.dat	
1839.9	8.8	1839.9	8.8	1839.9	8.8			Beta	63.9	Case_A_022.dat	
1311.4	207.0	1311.4	207.0	1311.4	207.0			Mode, µ	-15.5	Case_A_023.dat	
1671.6	7.9	1671.6	7.9	1671.6	7.9		Ex	pected Min.	-85.7	Case_A_024.dat	
1844.1	4.0	1844.1	4.0	1844.1	4.0	Raw Max	2244.1	Max		Case_A_025.dat	5
1700.1	6.1	1700.1	6.1	1700.1	6.1	Raw Min	2.3	Median	1720.1	Case_A_026.dat	
1477.2	120.3	1477.2	120.3	1477.2	120.3	Fltd Max	1844.1	StdDev	193.5	Case_A_027.dat	
1356.7	105.9	1356.7	105.9	1356.7	105.9	Fltd Min	4.0	Beta	150.9	Case_A_028.dat	
1824.4	20.6	1824.4	20.6	1824.4	20.6	Ave Max	1660.8	Mode, µ	1664.8	Case_A_029.dat	
1838.8	9.6	1838.8	9.6	1838.8	9.6	Ave Min	74.1	Ex Max.	2113.0	Case_A_030.dat	
1354.0	141.5	1354.0	141.5	1354.0	141.5			Min		Case_A_031.dat	
2244.1	3.7		3.7					Median	60.3	Case_A_032.dat	
1740.2	224.7	1740.2	224.7	1740.2	224.7	PE	95 %	StdDev	75.9	Case_A_033.dat	
1683.5	99.9	1683.5	99.9	1683.5	99.9			Beta	59.2	Case_A_034.dat	
2080.7	2.3	2080.7						Mode, µ	38.6	Case_A_035.dat	
1788.9	8.8	1788.9	8.8	1788.9	8.8		Exp	pected Min.	-26.4	Case_A_036.dat	
1642.6	286.4	1642.6	286.4	1642.6	286.4	Raw Max	1967.4	Max		Case_A_037.dat	6
1000.4	533.0	1000.4	533.0	1000.4	533.0	Raw Min	7.4	Median	1429.4	Case_A_038.dat	
1576.0	256.3	1576.0	256.3	1576.0	256.3	Fltd Max	1642.6	StdDev	238.3	Case_A_039.dat	
1297.5	327.7	1297.5	327.7	1297.5	327.7	Fltd Min	115.4	Beta	185.8	Case_A_040.dat	
1608.9	129.6	1608.9	129.6	1608.9	129.6	Ave Max	1363.5	Mode, µ	1361.3	Case_A_041.dat	
1052.7	439.2	1052.7	439.2	1052.7	439.2	Ave Min	292.1	Ex Max.	1913.2	Case_A_042.dat	
1786.5	7.4	1786.5						Min		Case_A_043.dat	
1490.7	172.1	1490.7	172.1	1490.7	172.1			Median	271.3	Case_A_044.dat	
1107.3	465.6	1107.3	465.6	1107.3	465.6	PE	95 %	StdDev	146.7	Case_A_045.dat	
1381.7	195.3	1381.7	195.3	1381.7	195.3			Beta	114.4	Case_A_046.dat	
1967.4	9.4		9.4					Mode, µ	229.4	Case_A_047.dat	
1477.1	115.4	1477.1	115.4	1477.1	115.4		Ex	pected Min.	103.9	Case_A_048.dat	

Appendix: Post-Processing Excel Spreadsheet Used in Analysis.

Crane Wire	Crane Wire	Delete	Delete	Fltd. Max	Fltd. Min	Statistica	l Post-				
Tension	Tension	Max.	Min.	Crane	Crane	Proces	sing	Gun	nble	file name ;	Tz
[max. KN]	[mm. KN]			Wire	Wire		Ŭ				
997.7	435.9	997.7	435.9	997.7	435.9	Raw Max	1740.0	Max		Case_A_049.dat	7
1499.0	294.1	1499.0	294.1	1499.0	294.1	Raw Min	9.0	Median	1241.8	Case_A_050.dat	
1239.1	377.6	1239.1	377.6	1239.1	377.6	Fltd Max	1499.0	StdDev	142.1	Case_A_051.dat	
1081.5	354.6	1081.5	354.6	1081.5	354.6	Fitd Mm	4/.4	Beta	110.8	Case_A_052.dat	
1309.8	4/.4	1309.8	4/.4	1309.8	47.4	Ave Max	1235.0	Mode, µ	1201.2	Case_A_055.dat	
1/40.0	258.2	1152.4	266.2	1152.4	251.5	Ave Min	297.8	LXMax	1550.4	Case_A_055 dat	
1244.5	452.7	1244.5	452.7	1244.5	452.7			Median	3101	Case A 056 dat	
1359.8	136.0	1359.8	136.0	1359.8	136.0	PE	95 %	StdDev	127.4	Case A 057 dat	
1278.6	344.1	1278.6	344.1	1278.6	344.1			Beta	99.3	Case A 058.dat	
1591.7	9.0	1591.7						Mode, µ	282.7	Case_A_059.dat	
1187.3	284.0	1187.3	284.0	1187.3	284.0		Eq	pected Min.	173.7	Case_A_060.dat	
994.4	432.5	994.4	432.5	994.4	432.5	Raw Max	1613.8	Max		Case_A_061.dat	8
1578.4	247.7	1578.4	247.7	1578.4	247.7	Raw Min	23.8	Median	1073.1	Case_A_062.dat	
1502.8	253.5	1502.8	253.5	1502.8	253.5	Fitd Max	1578.4	StdDev	208.6	Case_A_063.dat	
1118.7	225.5	1118.7	225.5	1118.7	225.5	Fltd Min	225.5	Beta	162.6	Case_A_064.dat	
991.7	533.5	991.7	533.5	991.7	533.5	Ave Max	1139.1	Mode, µ	1013.5	Case_A_065.dat	
1053.0	463.2	1053.0	463.2	1053.0	463.2	Ave Min	395.2	Ex Max	1496.6	Case_A_066.dat	
1073.1	339.7	1073.1	339.7	1073.1	339.7			Min		Case_A_067.dat	
1171.2	406.7	1171.2	406.7	1171.2	406.7			Median	432.5	Case_A_068.dat	
1087.9	496.5	1087.9	496.5	1087.9	496.5	PE	95 %	StdDev	111.2	Case_A_069.dat	
997.5	443.2	997.5	443.2	997.5	443.2			Beta	86.7	Case_A_070.dat	
962.0	505.2	962.0	505.2	962.0	505.2			Mode, µ	400.7	Case_A_071.dat	
1613.8	23.8						Eq	pected Min.	305.6	Case_A_072.dat	
1000.5	530.4	1000.5	530.4	1000.5	530.4	Raw Max	1300.9	Max		Case_A_073.dat	9
1072.3	474.4	1072.3	474.4	1072.3	474.4	Raw Min	110.9	Median	1036.4	Case_A_074.dat	
1300.9	403.3	1122.6	403.3	1122.6	255.0	Fitd Max	1237.1	StdDev	107.9	Case_A_0/5.dat	
1155.0	255.0	1133.0	200.0	1155.0	255.0	Fitd Mm	215.5	Deta Mode u	84.1 1005 5	Case_A_0/0.dat	
995.9	495.8	990.9	495.8	993.9	495.8	Ave Max	420.1	F.M.	1005.5	Case_A_0/7.aat	
978.8	400.7	9/0.0	400.7	9/8.8	400.7	Avemin	450.1	LX Max	1255.4	Case_A_070.dat	
073.3	576.5	073.3	576.5	003.3	576.5			Madian	470.6	Case_A_080 dat	
954.6	484.0	954.6	484.0	954.6	484.0	PF	95.%	StdDay	118.5	Case A 081 dat	
1237.1	215.5	1237.1	215.5	12371	215.5	12	22.70	Beta	92.4	Case A 082 dat	
1186.0	346.0	1186.0	346.0	1186.0	346.0			Mode. u	436.7	Case A 083 dat	
1198.2	110.9	1198.2	5 10.0	1100.0	510.0		Ea	pected Min.	335.3	Case A 084.dat	
907.6	599.3	907.6	599.3	907.6	599.3	Raw Max	1214.1	Max		Case A 085.dat	10
1149.2	354.6	1149.2				Raw Min	354.6	Median	1014.8	Case A 086.dat	
962.7	498.8	962.7	498.8	962.7	498.8	Fltd Max	1068.8	StdDev	56.9	Case_A_087.dat	
918.1	481.6	918.1	481.6	918.1	481.6	Fltd Min	400.1	Beta	44.4	Case_A_088.dat	
1042.9	632.6	1042.9	632.6	1042.9	632.6	Ave Max	995.0	$Mode, \mu$	998.6	Case_A_089.dat	
1214.1	502.0		502.0			Ave Min	522.0	Ex Max	1130.4	Case_A_090.dat	
1068.8	400.1	1068.8	400.1	1068.8	400.1			Min		Case_A_091.dat	
1021.2	470.4	1021.2	470.4	1021.2	470.4			Median	508.8	Case_A_092.dat	
1044.7	525.8	1044.7	525.8	1044.7	525.8	PE	95 %	StdDev	70.7	Case_A_093.dat	
1028.8	492.3	1028.8	492.3	1028.8	492.3			Beta	55.1	Case_A_094.dat	
1008.4	518.8	1008.4	518.8	1008.4	518.8			Mode, µ	488.6	Case_A_095.dat	
946.4	600.3	946.4	600.3	946.4	600.3		Eq	pected Min.	428.2	Case_A_096.dat	
1001.1	628.6	1001.1	628.6	1001.1	628.6	Raw Max	1131.7	Max		Case_A_097.dat	11
977.8	501.8	977.8	501.8	977.8	501.8	Raw Min	427.4	Median	997.0	Case_A_098.dat	
997.0	497.3	997.0	497.3	997.0	497.3	Fitd Max	10/1.2	StdDev	37.6	Case_A_099.dat	
941.8	580.5	941.8	580.5	941.8	580.5	Fitd Min	486.5	Beta	29.3	Case_A_0100.da	
1049.1	2.800	1049.1	558.5	1049.1	558.5	Ave Max	554.5	ivioαe, μ	986.3	Case_A_0101.da	
9/6.9	500.5	9/6.9	500.5	9/6.9	500.5	Ave Mm	554.5	Lx Max	1073.3	Case_A_0102.da	
000.2	500.5	900.2	590.5	999.2	590.5			Madian	559.5	Case A 0104 da	
10/3.0	192.1	1043.0	496.5	1043.0	496.5	DF	95.9/	StdDay	51.0	Case 4 0105 da	
980.0	400.5	980.0	400.5	980.0	400.5	TL I	95 /6	Bata	30.9	Case A 0106 da	
1131 7	427.7	200.0	-91.1	200.0	-21.1			Mode u	543.0	Case A 0107 da	
996.9	611.6	996.9	611.6	996.9	611.6		Ea	pected Min.	500.2	Case_A_0108.da	

Crane Wire Tension [max. kN]	Crane Wire Tension [min. kN]	Delete Max.	Delete Min.	Fltd. Max Crane Wire	Fltd. Min Crane Wire	Statistical Post- Processing		Gumble		file name ;	Tz
914.7	648.8	914.7	648.8	914.7	648.8	Raw Max	1155.3	Max		Case_A_0109.da	12
1133.1	434.3	1133.1				Raw Min	434.3	Median	952.6	Case_A_0110.da	
995.6	596.8	995.6	596.8	995.6	596.8	Fltd Max	1098.5	StdDev	72.3	Case_A_0111.da	
1034.8	445.5	1034.8	445.5	1034.8	445.5	Fltd Min	445.5	Beta	56.4	Case_A_0112.da	
1045.1	538.9	1045.1	538.9	1045.1	538.9	Ave Max	968.7	Mode, µ	931.9	Case_A_0113.da	
1098.5	470.7	1098.5	470.7	1098.5	470.7	Ave Min	551.6	ExMax	1099.3	Case_A_0114.da	
878.5	605.0	878.5	605.0	878.5	605.0			Min		Case_A_0115.da	
949.7	585.1	949.7	585.1	949.7	585.1			Median	555.6	Case_A_0116.da	
955.5	565.4	955.5	565.4	955.5	565.4	PE	95 %	StdDev	62.4	Case_A_0117.da	
901.8	545.8	901.8	545.8	901.8	545.8			Beta	48.7	Case_A_0118.da	
1155.3	505.3		505.3					Mode, µ	537.8	Case_A_0119.da	
912.3	514.0	912.3	514.0	912.3	514.0		Eq	pected Min.	484.4	Case_A_0120.da	
1111.6	453.7					Raw Max	1111.6	Max		Case_A_0121.da	13
975.9	633.8	975.9	633.8	975.9	633.8	Raw Min	453.7	Median	954.1	Case_A_0122.da	
954.1	592.1	954.1	592.1	954.1	592.1	Fltd Max	1054.3	StdDev	47.7	Case_A_0123.da	
934.2	570.6	934.2	570.6	934.2	570.6	Fltd Min	464.1	Beta	37.2	Case_A_0124.da	
980.8	630.3	980.8	630.3	980.8	630.3	Ave Max	956.1	Mode, µ	940.4	Case_A_0125.da	
1011.0	575.6	1011.0	575.6	1011.0	575.6	Ave Min	581.6	ExMax	1051.0	Case_A_0126.da	
912.1	546.6	912.1	546.6	912.1	546.6			Min		Case_A_0127.da	
899.3	553.9	899.3	553.9	899.3	553.9			Median	575.6	Case_A_0128.da	
920.2	625.7	920.2	625.7	920.2	625.7	PE	95 %	StdDev	53.7	Case_A_0129.da	
910.1	651.7	910.1	651.7	910.1	651.7			Beta	41.8	Case_A_0130.da	
965.4	553.7	965.4	553.7	965.4	553.7			Mode, µ	560.3	Case_A_0131.da	
1054.3	464.1	1054.3	464.1	1054.3	464.1		Eq	pected Min.	514.4	Case_A_0132.da	

SUMMARY OF RESULTS:

Tz	3	4	5	6	7	8	9	10	11	12	13
Raw Max	3388.6	1992.0	2244.1	1967.4	1740.0	1613.8	1300.9	1214.1	1131.7	1155.3	1111.6
Raw Min	0.0	0.6	2.3	7.4	9.0	23.8	110.9	354.6	427.4	434.3	453.7
Fltd Max	2054.5	1856.0	1844.1	1642.6	1499.0	1578.4	1237.1	1068.8	1071.2	1098.5	1054.3
Fltd Min	0.4	0.9	4.0	115.4	47.4	225.5	215.5	400.1	486.5	445.5	464.1
Ave Max	1684.6	1586.3	1660.8	1363.5	1235.0	1139.1	1063.2	995.0	1003.2	968.7	956.1
Ave Min	56.1	58.0	74.1	292.1	297.8	395.2	430.1	522.0	554.5	551.6	581.6
Exp. Max.	2108.2	2033.5	2113.0	1913.2	1530.4	1496.6	1255.4	1130.4	1073.3	1099.3	1051.0
Exp. Min.	-105.7	-85.7	-26.4	103.9	173.7	305.6	335.3	428.2	500.2	484.4	514.4

Note: .dat file under column filename are the simulations created on Orcaflex software.

Appendix: Abbreviation List

PHC:	Passive Heave Compensator
Hs:	Significant wave height
Tz:	Zero-up crossing period
Ψ:	Dynamic factor
SWL:	Safe Working Load
AHC:	Active Heave Compensator
Vc:	Hoisting velocity
Vr:	Relative velocity between load and hook at the time of pick up

<i>C</i> :	Spring constant or geometric stiffness coefficient referred to hook position (kN/m) $$
<i>g</i> :	Acceleration due to gravity
<i>W</i> :	Working load
Fd:	Hydraulic dampening force
α:	Discharge coefficient
A:	Flow area
Ak:	Compression area
ρ :	Oil density
Su:	Stroking velocity/piston rod velocity
<i>e</i> :	Efficiency of PHC
G(w):	Transmissibility
ωο:	Natural frequency
η3:	Vertical motion of lifted object
η <i>3T</i> :	Vertical motion at crane tip
RAO:	Response Amplitude Operator
PNE:	Probability of Non-Exceedance
kN:	kilo Newton
Fhyd:	Hydrodynamic force
Fmin.static:	Minimum static force
MBL:	Minimum Breaking Load

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